

Appl. No.: 10/523,564

Amdt. Dated March 24, 2006

Response to Office Action Mailed December 29, 2005

AMENDMENTS TO THE SPECIFICATION:

Please amend the paragraph beginning on page 7, line 11, of the specification as follows:

In Fig. 1, a movable frog denoted by 1 is displaceable into abutment on the wing rail 2 or 3, respectively. All of the components of the switching, locking and checking devices are arranged in a trough sleeper 4 below the track plane. The switching device 5 in this case is comprised of a cylinder piston unit and articulately connected to the stationary trough sleeper via bearings 6. As will be explained below, the bearings 6 are devised such that an adjustment of the switching device 5 is feasible in the longitudinal direction of the sleeper in the sense of double arrow 7 so as to enable the adjustment of the center position of the cylinder piston unit. The switching device 5 is coupled to a driver part 8 which transmits the switching movement to the movable frog 1. Driving is effected via first displaceable stops 9 cooperating with a sliding block 10, which in turn is connected with the base plate 11 of the movable frog 1. The precise adjustment of the effective stroke of the movable frog 1 is feasible by the adjustment of an idle stroke between ~~the~~ these first stops 9 and the sliding block 10.

Please amend the paragraph beginning on page 8, line 1, of the specification as follows:

Fig. 2 depicts the individual coupling and driver parts on an enlarged scale. It is apparent that the driver 8, which is pivotally supported on the cylinder piston unit, is comprised of two sleeves 34 surrounding the cylinder piston unit 5 and having projections 12 through which a spindle 13 passes, the position of the spindle 13 relative to the projections 12 of the driver 8 being fixed by stop shoulders formed by the larger diameter-region 14 of the spindle 13. The larger-diameter-region 14 of the spindle 13 further comprises two threaded portions 15 and 16 having mutually opposed thread directions. A turn of the spindle causes the first stop parts 9 mounted on the threaded regions 15 and 16 in a rotationally fast manner to be moved away from each other, or towards each other, in the manner of a driver nut in the sense of double arrow 17. The driver part further comprises a sliding block 10 through which the spindle 13 passes and on which it is mounted so as to be slidably movable between the first stops 9. By displacing the first stops 9 in the sense of double arrow 17, it is feasible to adjust the idle stroke a between the first stops 9 and the sliding block 10 in order to reduce the switching stroke of the cylinder piston unit 5 to the respectively required displacement stroke of the movable frog 1.

Please amend the paragraph beginning on page 9, line 4, of the specification as follows:

From the side view according to Fig. 3, the stirrup-shaped form of the driver part 21 is apparent, it being recognizable that the sliding block 10 is slidably movable in the stirrup-shaped driver part 21 along double arrow 22 such that longitudinal displacements of the

movable frog 1 possibly caused, for instance, by thermal expansions will not be transmitted to the switching mechanism. Furthermore, it is apparent that the first stops 9, i.e. the driver nuts, are supported on the sleeves 34 of the drivers in a rotationally fast manner. This is to obtain an anti-rotation lock of the first stops 9 relative to the rotation of the spindle and to ensure the axial displacement of the first stops 9. The positioning of the driver 8 relative to the axis 23 at a central angle α occurs as a function of the upward or downward movement in the sense of double arrow 24 and the displacement in the longitudinal direction of the rail in the sense of double arrow 22, of the movable frog such that forces resulting from these movements will be prevented from being introduced into the cylinder piston unit.

Please amend the paragraph beginning on page 9, line 24, of the specification as follows:

Fig. 4 elucidates the mounting of the cylinder piston unit 5 on the trough sleeper 4. In this case, the bearing 6 is provided with a fork head 25 whose fork is supported in a rotationally fast manner while being displaceable in the direction of the axis 23 of the cylinder piston unit 5, and is connected with the hydraulic cylinder piston unit 5 via a bearing journal 26. Here, the fixation of the rotary position of the fork head 25 is effected in that the fork head 25 is supported on a web 27 departing from the side wall of the trough sleeper. The fork head 25 is connected with a fork head screw 28 carrying an adjusting ~~serew~~ nut 29. A turn of the adjusting ~~serew~~ nut 29, whose axial position is fixed by the aid of the second stop 30 rigidly connected with the trough sleeper 4, causes an axial displacement of the fork head 25 in the sense of double arrow 31. The adjusted axial position of the fork head is fixed by the aid of union part 32 and

nut 33. This axial displacement of the fork head, and hence the cylinder piston unit, which must, of course, take place in both of the adjustable bearings 6 on both sides of the cylinder piston unit, enables the precise adjustment of the center position of the piston stroke.